A short course on

Structural Members

This video: St. Venant Torsion

Terje's Toolbox is freely available at <u>terje.civil.ubc.ca</u> It is created and maintained by Professor Terje Haukaas, Ph.D., P.Eng., Department of Civil Engineering, The University of British Columbia (UBC), Vancouver, Canada

Two Types of Torsion



Ingredients



Notation



x =longitudinal axis

- m_x = distributed torque along the member
- T =torque, resultant of shear stress
- ϕ = rotation about the *x*-axis
- J = cross-section constant for St. Venant torsion
- $G = \text{shear modulus} = E/(2(1+\nu))$
- τ = shear stress
- γ = shear strain

Equilibrium





Section Integration



$$T = \int_{r_i}^{r_o} (2\pi r) \cdot \tau \cdot r \cdot \mathrm{d}r$$

Material Law

$$\tau = G \cdot \gamma$$

$$G = \frac{E}{2(1+\nu)}$$

Kinematic Compatibility





Summary

$$J = \int_{r_i}^{r_o} 2\pi r^3 \, \mathrm{d}r = \frac{\pi}{2} \cdot \left(r_o^4 - r_i^4\right)$$



Prandtl's Stress Function



 $\tau_{xy} = \frac{\partial P}{\partial z} = P_{z}$ $\tau_{xz} = -\frac{\partial P}{\partial y} = -P_{y}$







Boundary Conditions for Stress Function



$$\tau_{xr} = 0 \qquad \qquad \tau_{xr} = \frac{\partial P(r,s)}{\partial s} = 0 \qquad \qquad \tau_{xr} = \tau_{xz} \cdot \cos(\alpha) - \tau_{xy} \cdot \sin(\alpha) = -\left(\frac{\partial P}{\partial y}\frac{\partial y}{\partial s} + \frac{\partial P}{\partial z}\frac{\partial z}{\partial s}\right) = \frac{\partial P}{\partial s} = 0$$

Membranes



Section Integration

$$T = \int_{A} \left(\tau_{xz} \cdot y - \tau_{xy} \cdot z \right) \mathrm{d}A$$

$$T = -\int_{A} \left(\frac{\partial P}{\partial y} \cdot y + \frac{\partial P}{\partial z} \cdot z \right) \mathrm{d}A$$

$$T = -\left(\int_{A} \frac{\partial P}{\partial y} \cdot y \cdot dA\right) - \left(\int_{A} \frac{\partial P}{\partial z} \cdot z \cdot dA\right)$$
$$= -\left(\oint P \cdot y \cdot d\Gamma - \int_{A} P \cdot dA\right) - \left(\oint P \cdot z \cdot d\Gamma - \int_{A} P \cdot dA\right)$$
$$= -\oint \frac{P \cdot y \cdot d\Gamma}{0} - \oint \frac{P \cdot z \cdot d\Gamma}{0} + \int_{A} P \cdot dA + \int_{A} P \cdot dA$$
$$= 2 \cdot \int_{A} P \cdot dA$$

Kinematic Compatibility

 $v = -\phi \cdot z$ $w = \phi \cdot y$

$$\varepsilon_x = \frac{du}{dx} = 0$$
$$\varepsilon_y = \frac{dv}{dy} = 0$$
$$\varepsilon_z = \frac{dw}{dz} = 0$$

$$\gamma_{xy} = \frac{dv}{dx} + \frac{du}{dy} = -\frac{d\phi}{dx} \cdot z + \frac{du}{dy}$$
$$\gamma_{xz} = \frac{du}{dz} + \frac{dw}{dx} = \frac{du}{dz} + \frac{d\phi}{dx} \cdot y$$
$$\gamma_{yz} = \frac{dw}{dy} + \frac{dv}{dz} = 0$$

Summary



 $P_{z} = \tau_{xy} = G \cdot \gamma_{xy} = G \cdot \left(-\phi_{,x} \cdot z + u_{,y}\right)$ $P_{y} = -\tau_{xz} = -G \cdot \gamma_{xz} = -G \cdot \left(u_{,z} + \phi_{,x} \cdot y\right)$ $\frac{\partial^{2} P(y,z)}{\partial y^{2}} + \frac{\partial^{2} P(y,z)}{\partial z^{2}} \equiv P_{,yy} + P_{,zz} \equiv \nabla^{2} P(y,z) = -2 \cdot G \cdot \phi'$

Cross-section Analysis

Cross-section constant, J

Shear stress τ_{xy} and τ_{xz}



More lectures:

Terje's Toobox:

terje.civil.ubc.ca